

## ROUGHNESS ON ROAD AND ON BRIDGE

Tassos P. Avraam and George T. Michaltsos

National Technical University of Athens, Athens, Greece  
e-mail: michalts@central.ntua.gr, avraamt@central.ntua.gr

**ABSTRACT:** A new point of view of how the surface roughness operates on the cars is the subject of this paper. The paper has two parts. The first studies the roughness on road while the second studies the roughness on the bridge deck. The theoretical formulation is based on a continuous approach that has been used in the literature to analyze such bridges, the procedure is carried out by the modal superposition method, while the obtained equations are solved using Duhamel's integrals.

**KEYWORDS:** Roughness; Road surface; Bridge deck roughness.

### 1 INTRODUCTION

The influence of a rough deck-surface on the dynamic response of a bridge depends on various factors. The irregularities, on a deck-surface, may be due to the roughness of the deck-surface, but also due to random or on purpose existing anomalies for traffic reasons. The parameter of the road surface (or bridge deck) roughness, dealt with excessively in the recent literature.

In 1960, Carey and Irick [1] showed that surface roughness was the primary variable needed to explain the driver's opinion of the quality of the serviceability provided by a pavement surface. Most of the followed investigations focused on the study, characterization and classification of the pavements [2 to 6].

Numerous studies adopt the Power Spectral Density (PSD) functions for roughness, as modified by Wang and Huang [7], or the simpler harmonically varying surface irregularity presented in Cheng et al [8].

Among these one most quote the significant contribution by Fafard et al [9], Cheng and Lee [10], Huang and Wang [11], Kou and DeWolf [12] as well as by Yang et al [13]. Their findings have shown that the foregoing parameter is one of the most important factors affecting elastic dynamic response, especially applicable to steel highway bridges.

The research of today is focused on two main subjects. A number of researchers are studying the noise and inconvenience caused by the pavement roughness [14, 15], or the percentage influence of roughness on the vehicle-structure coupled interaction [16, 17]. Other researchers are trying to determine

the bridges' frequencies by studying the influence of the road surface roughness on a moving vehicle or a pair of vehicles [18 to 20].

It is obvious that the wheel of a vehicle cannot follow the entire surface of the roughness because of the wheels' dimensions and the surface of the roughness. So the inconvenience caused should be sought for other reasons. Although some researchers have suspected and reported [21, 22] the effect of tire bounce, the study in this area did not continue further.

In this paper the problem is studied by another point of view. A possible reason could be the forces developing during the rolling of the wheels on the abnormal surface of the deck-road. The roughness is considered as a series of repeated small irregularities, which the wheel passes without changing its level of motion but with the development of impact forces from each small irregularity (fig. 2).

The paper has two parts. The first studies the roughness on road while the second studies the roughness on the bridge deck. The theoretical formulation is based on a continuous approach that has been used in the literature to analyze such bridges, the procedure is carried out by the modal superposition method, while the obtained equations are solved using Duhamel's integrals.

A variety of numerical examples allows one to draw important conclusions for structural design purpose.

## 2 INTRODUCTORY CONCEPTS

When the solid bodies shown in figure A1, with rotational velocities  $\theta_1$  and  $\theta_2$  and rotational inertias  $I_1$  and  $I_2$  are collided, after their bouncing, the new rotational velocity of the first body after impact is given by:

$$\theta_{1a} = \frac{(I_1 - \varepsilon I_2)\theta_{1f} + (1 + \varepsilon)I_2\theta_{2f}}{I_1 + I_2} \quad (1a)$$

where  $\varepsilon$  is the percentage of the energy remaining after impact (which for regular wheels is between 0.80 and 0.97, depending on the inner pressure of the wheel) and f and a express the rotational velocities before and after impact.

Setting into (1a)  $I_1 = \infty$  and  $\theta_{1f} = 0$ , we obtain:

$$\theta_{2a} = -\varepsilon\theta_{2f} = -\varepsilon\theta \quad (1b)$$

A body under the action of a moment for a period of time  $\Delta t$ , acquires a torsional momentum (or impulse) equal to:

$$G = M \cdot \Delta t \quad (2a)$$

It is proved that:

$$G = I \cdot \omega \quad (2b)$$

where I the rotational inertia of the body and  $\omega$  its rotational velocity.

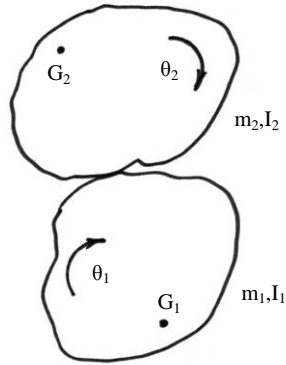


Figure 1. Collision of two bodies

A wheel, has rotational inertia  $I = m_o i^2$  or finally

$$I = m_o R^2 / 2 \tag{3}$$

where  $m_o$  is the mass of the wheel,  $i$  its inertia radius and  $R$  the radius of the wheel.

According to appendix A, the impact of a wheel with radius  $R$  on a solid plate is expressed by

$$G = M \cdot \Delta t = N \cdot \Delta t \cdot R = I \cdot \omega = m_o \varepsilon R^2 \omega = m_o \varepsilon \cdot \frac{R^2}{2} \cdot \frac{v}{R}$$

and finally 
$$N = G_{\Delta t \rightarrow 0} = \varepsilon \cdot m_o v / 2 \tag{4}$$

where  $N$  is the force arising from the impact and acting on the common tangent of two solids at the point of impact (see also Figure 2).

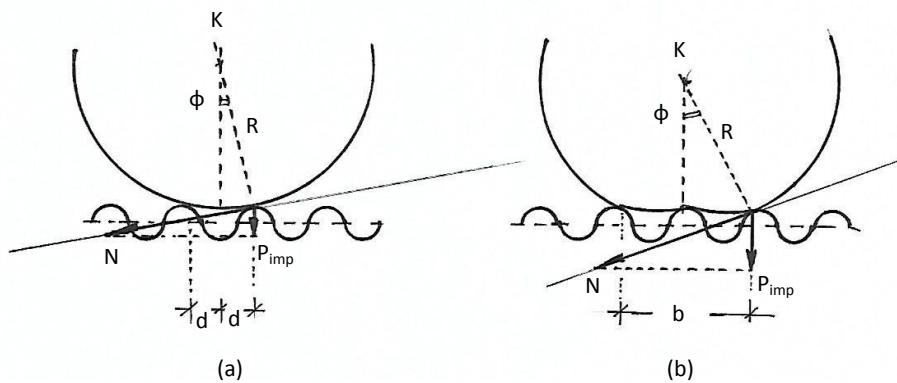


Figure 2. Impact because of irregularities

### 3 EQUATIONS OF THE PROBLEM

#### 3.1 Roughness on road

The acting on the road force  $F$  is:

$$F = (M + m_o)g - M\ddot{z} + P_{imp} \quad (5a)$$

Cutting at a-a and taking the equilibrium of the cut off part we have:

$$F = kz + c\dot{z} \quad (5b)$$

From (5a) and (5b) we obtain the following equation of the problem:

$$M\ddot{z} + c\dot{z} + kz = (M + m_o)g + P_{imp} \quad (5c)$$

But

$$P_{imp} = N \cdot \sin \varphi = N \cdot \frac{d}{R} \quad (5d)$$

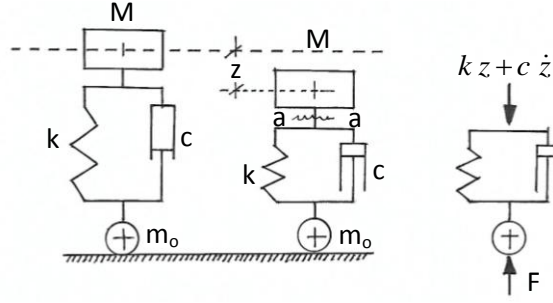


Figure 3. Mass load on road

Equation (5c), because of (4) and (5d) becomes:

$$\left. \begin{aligned} M\ddot{z} + c\dot{z} + kz &= (M + m_o)g + \frac{\varepsilon m_o \nu d}{2R} \cdot \delta(t - t_n) \\ \text{where: } t_n &= (n-1) \frac{2d}{\nu} \end{aligned} \right\} \quad (5e)$$

Where  $\delta$  is the Dirac Delta function and  $n$  the  $n^{\text{th}}$  irregularity of the roughness. The above (5e) can be written as follows:

$$\left. \begin{aligned} \ddot{z} + 2\beta\dot{z} + \omega^2 z &= \frac{(M + m_o)g}{M} + \frac{\varepsilon m_o \nu d}{2RM} \cdot \delta(t - t_n) \\ \text{where: } 2\beta &= \frac{c}{M}, \quad \omega^2 = \frac{k}{M} \end{aligned} \right\} \quad (6)$$

The first term of the right side member of equation (6) gives the time needed to tranquillize the system in its equilibrium state after the application of mass  $M$ . The second term expresses the roughness influence. Solving equation (6) without the term due to roughness we obtain:

$$z_s(t) = \frac{(M + m_o)g}{\omega_o M} \int_0^t e^{-\beta(t-\tau)} \sin \omega_o(t-\tau) d\tau = \frac{(M + m_o)g}{\omega_o M} \left[ \omega_o - e^{-\beta t} (\beta \sin \omega_o t + \omega_o \cos \omega_o t) \right]$$

where:  $\omega_o = \sqrt{\omega^2 - \beta^2}$

(7a)

Putting for example  $M = 300 \text{ kg}$ ,  $m_o = 8 \text{ kg}$ ,  $k = 60000 \text{ dN/m}$ ,  $c = 1000 \text{ dN sec/m}$  and applying equation (7), we get the plot of figure 4.

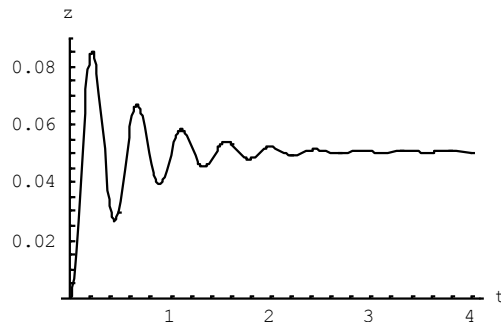


Figure 4. Tranquillization of system

We observe that after a needed time (in this example it is  $\sim 2.5$  sec), the system tranquillizes after equilibrium at  $z = 5 \text{ cm}$ . This verifies the equation  $z = Mg/k = 3000/60000 = 0.05 \text{ m}$ , which gives the position of the static equilibrium.

We suppose that at  $t=0$  the vehicle has already taken its static equilibrium position and starts to move with speed  $v$ , on a road with roughness. The equation of motion is:

$$\ddot{z} + 2\beta \dot{z} + \omega^2 z = \frac{\varepsilon m_o v d}{2RM} \cdot \delta(t - t_n)$$

where:  $t_n = (n-1) \frac{2d}{v}$

(7b)

When the load moves from irregularity (n-1) to irregularity (n), the solution of equation (7b) is given by the Duhamel's integral:

$$z_{n-1}(t) = \frac{\varepsilon m_o v d}{2\omega_o RM} \int_0^t e^{-\beta(t-\tau)} \sin \omega_o(t-\tau) \delta(\tau - t_{n-1}) d\tau$$

or finally:

$$z_{n-1}(t) = \frac{\varepsilon m_o v d}{2\omega_o RM} \cdot e^{-\beta(t-t_{n-1})} \sin \omega_o(t-t_{n-1}) [H(t-t_{n-1}) - H(t-t_n)] \quad (7c)$$

where H is the Haeviside unit step function.

When the load moves from irregularity (n) to (n+1) one, the solution of equation (7d) is:

$$\left. \begin{aligned} z_n(t) = e^{-\beta(t-t_n)} [A \sin \omega_o(t-t_n) + B \cos \omega_o(t-t_n)] \cdot [H(t-t_n) - H(t-t_{n+1})] \\ + \frac{\varepsilon m_o \nu d}{2 \omega_o R M} \cdot e^{-\beta(t-t_n)} \sin \omega_o(t-t_n) \cdot [H(t-t_n) - H(t-t_{n+1})] \end{aligned} \right\} \quad (7d)$$

With time conditions  $z_{n-1}(t_n) = z_n(t_n)$ ,  $\dot{z}_{n-1}(t_n) = \dot{z}_n(t_n)$  we determine the constants A and B as follows:

$$\left. \begin{aligned} A &= \frac{\varepsilon m_o \nu d}{2 \omega_o R M} \cdot e^{-\beta(t_n-t_{n-1})} \cos \omega_o(t_n - t_{n-1}) \\ B &= \frac{\varepsilon m_o \nu d}{2 \omega_o R M} \cdot e^{-\beta(t_n-t_{n-1})} \sin \omega_o(t_n - t_{n-1}) \end{aligned} \right\} \quad (8a)$$

In the above equations, the term  $L = e^{-\beta(t_n-t_{n-1})}$  has limit  $\rightarrow 1$ . This is clear in the plot of figure 5 (for  $M = 300 \text{ kg}$ ,  $m_o = 8 \text{ kg}$ ,  $k = 60000 \text{ dN/m}$ ,  $c = 1000 \text{ dN sec/m}$ ), different vehicle's speeds and roughness quality.

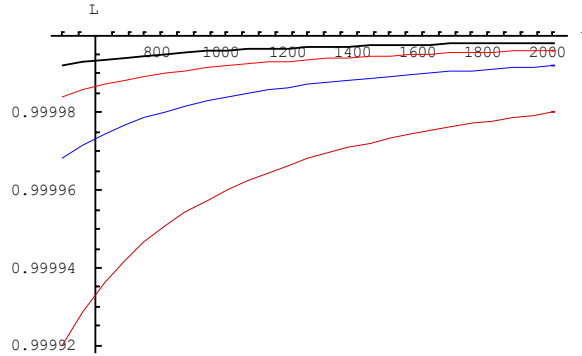


Figure 5.  $d=0.01$ (black),  $d=0.02$ (red),  $d=0.04$ (blue),  $d=0.10$  (café)

Therefore one can write:

$$\left. \begin{aligned} A &= \frac{\varepsilon m_o \nu d}{2 \omega_o R M} \cdot \cos \omega_o(t_n - t_{n-1}) \\ B &= \frac{\varepsilon m_o \nu d}{2 \omega_o R M} \cdot \sin \omega_o(t_n - t_{n-1}) \end{aligned} \right\} \quad (8b)$$

### 3.2 Roughness on bridge

Easily one can find that:

$$P_{imp} = \frac{\varepsilon m_o v}{2} \cdot \sin \varphi = \frac{\varepsilon m_o v}{2} \cdot \frac{d}{R} \quad (9)$$

with  $d$ , determined according to ISO 8608 and 4287.

Figure 2a assumes that the wheel will be undeformed. Otherwise, figure 2b applies when  $b > 3d$ . In the first case it will be  $\tan \varphi = d/R$  while in the second one  $\tan \varphi = b/2R$ .

Assuming that two successive elevations of roughness are  $2d$  apart, then when the wheel impacts on the  $a_i^{th}$  elevation, the mathematical formulation will be:

$$G = P_{imp} \cdot \delta(x - a_i) \cdot \delta(t - t_{a_i}) \quad (10)$$

where  $\alpha_i$  is the distance of the  $i^{th}$  elevation from the left end of the bridge.

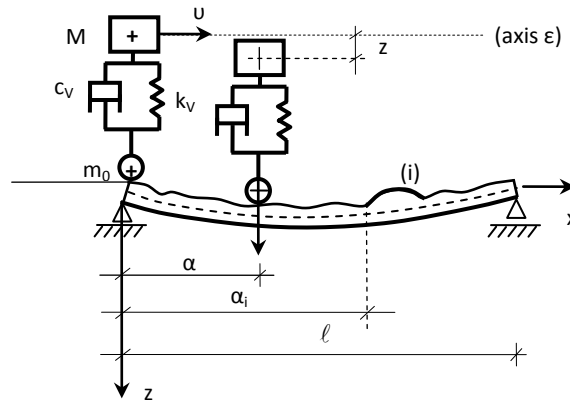


Figure 6. Mass load on bridge

Therefore the corresponding equations of bridge's motion are:

$$\left. \begin{aligned} EI_y w'''' + c_b \dot{w} + m \ddot{w} &= [(M + m_o)g - M \ddot{z} - m_o \ddot{w}] \cdot \delta(x - a) + P_{imp} \sum_{i=1}^{\zeta} \delta(x - a_i) \cdot \delta(t - t_{a_i}) \\ M \ddot{z} + c_p \dot{z} + k_p z &= k_p w(x) + c_p \dot{w}(x) \quad , \text{with } \zeta = L/2d, t_{a_i} = i \cdot 2d/v \end{aligned} \right\} \quad (11a)$$

Solving the second of equations (11a) we conclude to the following expressions:

$$\begin{aligned}
\ddot{z}(t) &= \Phi(t) + \frac{1}{\bar{\omega}_p} \int_0^t \Xi(\tau) \cdot \Phi(\tau) d\tau \\
\text{where: } \Phi(t) &= \omega_p^2 w(a, t) + 2\beta_p \dot{w}(a, t) \\
\Xi(t) &= e^{-\beta_p(t-\tau)} [(\beta_p^2 - \bar{\omega}_p^2) \sin \bar{\omega}_p(t-\tau) + 2\beta_p \bar{\omega}_p \cos \bar{\omega}_p(t-\tau)] \\
\beta_p &= c_p / 2M, \quad \omega_p^2 = k_p / M, \quad \bar{\omega}_p = \sqrt{\omega_p^2 - \beta_p^2}
\end{aligned} \quad \left. \vphantom{\ddot{z}(t)} \right\} (11b)$$

In order to solve the first of equations (11a), we are searching for a solution of the form:

$$w(x, t) = \sum_n X_n(x) \cdot T_n(t) \quad (11c)$$

where  $X_n$  are the shape functions of the beam and  $T_n$  are the time functions to be determined. Following the well known procedure and using approaching methods to solve the equations governing the motion of the bridge [23], we finally find:

$$\begin{aligned}
T_k(t) &= \frac{2}{mL\bar{\omega}_k} \int_0^t [(M + m_o)g - M \ddot{z}(\tau) - m_o \sum_n \sin \Omega_n \tau \ddot{T}_n(\tau)] \sin \Omega_k \tau \cdot e^{-\beta(t-\tau)} \sin \bar{\omega}_k(t-\tau) d\tau \\
&\quad + \frac{\varepsilon m_o v d}{mLR\bar{\omega}_k} \sum_{i=1}^{\zeta} \sin \Omega_k t_a \cdot e^{-\beta(t-t_a)} \sin \bar{\omega}_k(t-t_a) H(t-t_a)
\end{aligned} \quad \left. \vphantom{T_k(t)} \right\} (12a)$$

where  $\zeta = L/2d$ ,  $t_a = i \cdot 2d/v$ .

After the impact of the wheel on the  $i^{\text{th}}$  irregularity of the roughness, its influence lasts until its impact on the  $(i+1)^{\text{th}}$  one. Then at  $t_{i+1} = t_i + 2d/v$ , the beam vibrates under the action of  $P_{imp}$  on  $(i+1)^{\text{th}}$  irregularity and under the appeared free motion due to the previous action of  $P_{imp}$  on the  $i^{\text{th}}$  irregularity. This last term for the  $k$  mode has the following time function:

$$T_k^{free}(t) = \frac{2P_{imp}}{mL\omega_k} \cdot e^{-\beta(t-t_i)} [A_k \sin \omega_k(t-t_i) + B_k \cos \omega_k(t-t_i)] \quad (12b)$$

Using the time conditions

$$T_k(t_i) = T_k^{free}(t_i), \quad \text{and} \quad \dot{T}_k(t_i) = \dot{T}_k^{free}(t_i) \quad \left. \vphantom{T_k(t_i)} \right\} (12c)$$

We finally find:

$$\begin{aligned}
A_k &= e^{-\beta(t_i-t_{i-1})} \cdot \sin \Omega_k t_{i-1} \cdot \cos \omega_k(t_i - t_{i-1}) \\
B_k &= e^{-\beta(t_i-t_{i-1})} \cdot \sin \Omega_k t_{i-1} \cdot \sin \omega_k(t_i - t_{i-1})
\end{aligned} \quad \left. \vphantom{A_k} \right\} (12d)$$



Let us now consider the vehicle of figure 7.

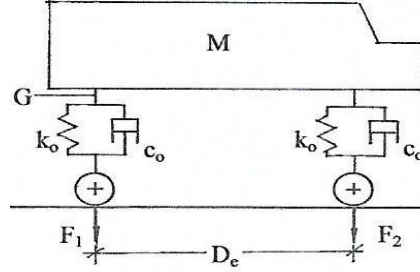


Figure 7. Vehicle's model

Assuming that the weight of the vehicle is equally distributed between the front and back wheels, the forces applied on the bridge will be:

$$\left. \begin{aligned} F_1 &= \left[ \frac{M}{2} (g - \ddot{z}) + m_o (g - \ddot{w}) \right] \delta(x - a) + \frac{\varepsilon m_o v}{2} \sin \varphi \sum_{i=1}^{\xi} \delta(x - a) \delta(t - t_a) \\ F_2 &= \left[ \frac{M}{2} (g - \ddot{z}) + m_o (g - \ddot{w}) \right] \delta(x - a + D_e) + \frac{\varepsilon m_o v}{2} \sin \varphi \sum_{i=1}^{\xi} \delta(x - a + D_e) \delta(t - t_a) \end{aligned} \right\} \quad (13a)$$

where  $\ddot{z}$  from equation (11d), and  $D_e$  the wheelbase of the vehicle. Searching for a solution under the form:

$$w(x, t) = \sum_n X_n(x) \cdot T_n(t) \quad (13b)$$

we determine the time function  $T_k(t)$  as follows:

$$\left. \begin{aligned} T_k &= \frac{2}{mL\bar{\omega}_k} \int_0^t G_1(\tau) e^{-\beta(t-\tau)} \sin \bar{\omega}_k(t-\tau) d\tau + \frac{\varepsilon m_o v}{2m\bar{\omega}_k} \sin \varphi \sum_{i=1}^{\xi} \sin \Omega_k t_{a1} e^{-\beta(t-t_{a1})} \sin \bar{\omega}_k(t-t_{a1}) H(t-t_{a1}) \\ &+ \frac{2}{mL\bar{\omega}_k} \int_0^t G_2(\tau) e^{-\beta(t-\tau)} \sin \bar{\omega}_k(t-\tau) d\tau + \frac{\varepsilon m_o v}{2m\bar{\omega}_k} \sin \varphi \sum_{i=1}^{\xi} \sin \Omega_k t_{a2} e^{-\beta(t-t_{a2})} \sin \bar{\omega}_k(t-t_{a2}) H(t-t_{a2}) \end{aligned} \right\} \quad (13c)$$

where:

$$\left. \begin{aligned} G_1(t) &= \left[ \left( \frac{M}{2} + m_o \right) g - \frac{M}{2} \ddot{z}(t) - m_o \sum_n \sin \Omega_n t_1 \ddot{T}_n(t) \right] \sin \Omega_k t_1 \\ G_2(t) &= \left[ \left( \frac{M}{2} + m_o \right) g - \frac{M}{2} \ddot{z}(t) - m_o \sum_n \sin \Omega_n t_2 \ddot{T}_n(t) \right] \sin \Omega_k t_2 \\ t_{a1} &= \frac{\delta_i + \xi_o}{v}, \quad t_{a2} = \frac{\delta_i + \xi_o - D_e}{v}, \quad t_1 = \frac{\delta_i}{v}, \quad t_2 = \frac{\delta_i - D_e}{v} \end{aligned} \right\} \quad (13d)$$

#### 4 NUMERICAL EXAMPLES AND DISCUSSION

The purpose of this paper is to study the effect of the impact of the vehicle's tires on the roughness' irregularities for a vehicle moving either on the road or on a bridge.

The paper focuses on the following parameters:

- The quality of the road surface and its effect on the so called passengers' trouble.
- The effect of the roughness on the bridge motion

For the study of the road surface we use the vehicle of figure 3, while for the study of the bridges we use the vehicle of figure 7.

The used vehicle, has the following characteristics:

$$M = 300 \text{ kg} , m_o = 8 \text{ kg} , R = 0.30 \text{ m} , k_o = 60000 \text{ dN} / \text{m} , c_o = 1000 \text{ dN} \cdot \text{sec} / \text{m} .$$

The vehicles' speed is  $v = 10 \text{ m} / \text{sec} .$

As for the roughness study three kinds of pavement are considered. The soft with  $d = 0.01 \text{ m}$ , the middle with  $d = 0.02 \text{ m}$ , and the bad with  $d = 0.035 \text{ m}$ . Finally the case of a cobbled road is studied with  $d = 0.10 \text{ m}$ .

The bridges are made from homogeneous and isotropic material, having modulus of elasticity  $E = 2.1 \cdot 10^{10} \text{ dN} / \text{m}^2 .$

Let us consider two kinds of bridges.

- One, relatively short bridge of length  $L = 20 \text{ m}$ , mass per unit length  $m = 250 \text{ kg} / \text{m}$ , and moment of inertia  $I_y = 0.01 \text{ m}^4 .$

Note should be taken of the following:

- The vehicles are supposed to move along the center line of the bridge. Thus no rotational motion is developed.
- They are studied the displacements of the bridge middle of the span.
- Only the first six flexural modes are taken into account.

##### 4.1 Roughness on road

Applying the formulae of §3.1 and considering that the system of figure 3 is tranquilized in its equilibrium position, we obtain the following plots.

In the plot of figure 8 we see the motion of the vehicle for a smooth road surface ( $d=0.01\text{m}$ ). The so produced trouble is the result of the vehicle motion from  $0.000016 \text{ m}$  to  $0.000024 \text{ m}$  or for  $0.000008 \text{ m}$  which can be considered as practically non-existent.

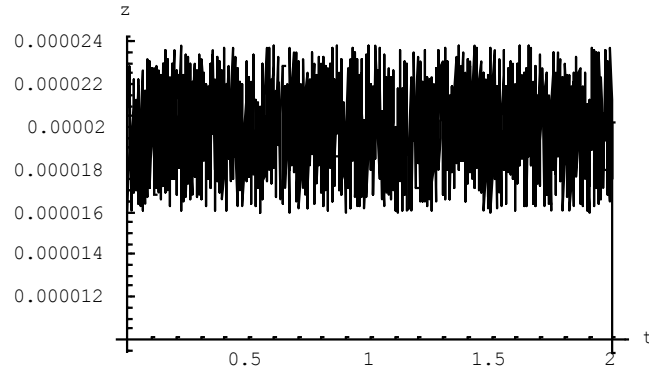


Figure 8. Roughness on road with  $d=0.01\text{m}$

In the plot of figure 9 we see the motion of the vehicle for a middle road surface ( $d=0.02\text{m}$ ). The so produced trouble is the result of the vehicle motion from  $0.000063\text{ m}$  to  $0.000097\text{ m}$  or for  $0.000034\text{ m}$ , which causes a rather sufferable trouble.

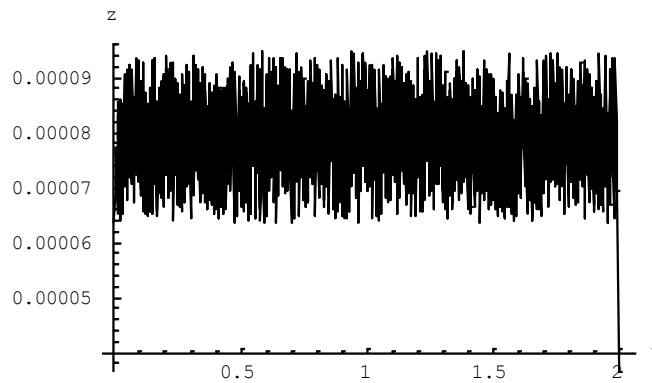


Figure 9. Roughness on road with  $d=0.02\text{m}$

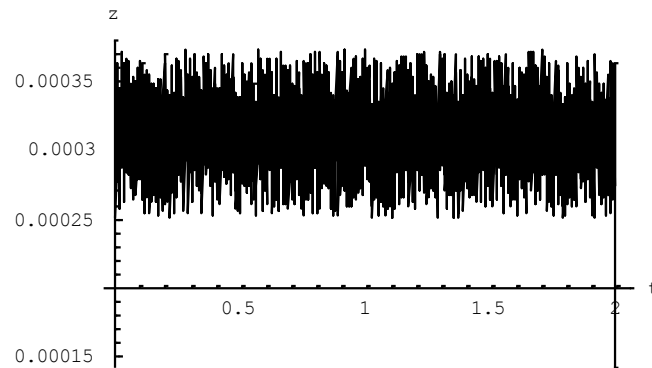


Figure 10. Roughness on road with  $d=0.04\text{m}$

In the plot of figure 10 we see the motion of the vehicle for a bad road surface ( $d=0.04m$ ). The so produced trouble is the result of the vehicle motion from  $0.00043 m$  to  $0.00024 m$  or for  $0.00019 m$ , which causes a notable trouble.

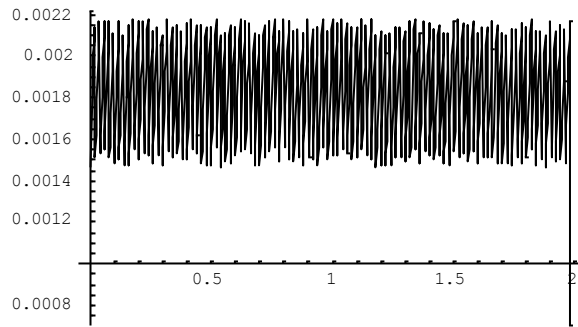


Figure 11. Roughness on road with  $d=0.10m$

Finally in the plot of figure 11 we see the motion of the vehicle for a very bad road surface ( $d=0.10m$  – cobbled road). The so produced trouble is the result of the vehicle motion from  $0.00132 m$  to  $0.00233 m$  or for  $0.00101 m$ , which causes an annoying trouble.

### 4.2 Roughness on bridge

Let us consider now two road surface qualities, the bad road surface ( $d \leq 0.04m$ ), and a surface with cobbled road ( $d = 0.10m$ ).

Applying the formulae of paragraph 3.2, we obtain the following diagrams related to the motion of the middle of the bridge for a vehicle moving with speed  $v = 10 m/sec$ , and different kind of roughness.

In the following figure 12 we see the plots showing the motion of the middle of the bridge due to the passage of a light vehicle moving with speed  $10 m/sec$  on a road surface with bad roughness ( $d = 0.04m$ ).

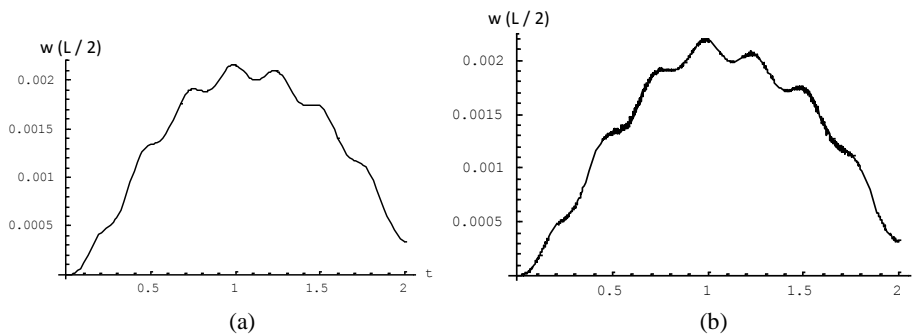


Figure 12. a) Bridge without roughness b) Bridge with roughness

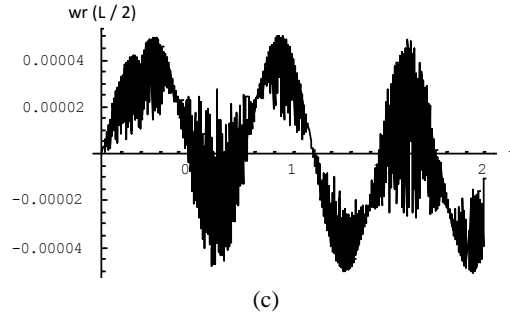


Figure 12. c) Deformation of the bridge due to roughness only

In the following figure 13 we see the plots showing the motion of the middle of the bridge due to the passage of a light vehicle moving with speed 10 m/sec on a surface with cobbled road ( $d = 0.10\text{ m}$ ).

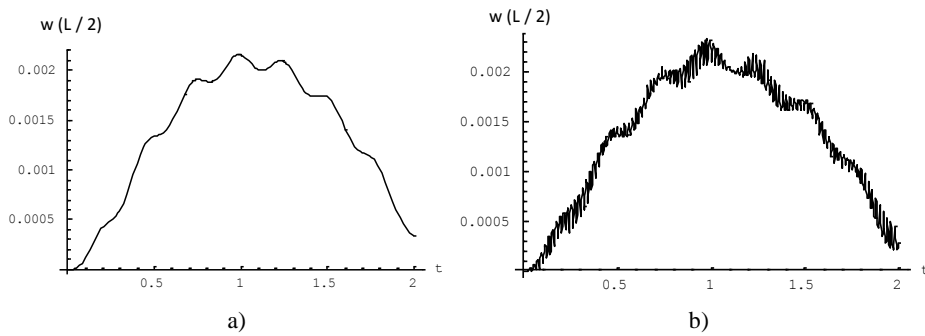


Figure 13. a) Bridge without roughness b) Bridge with roughness

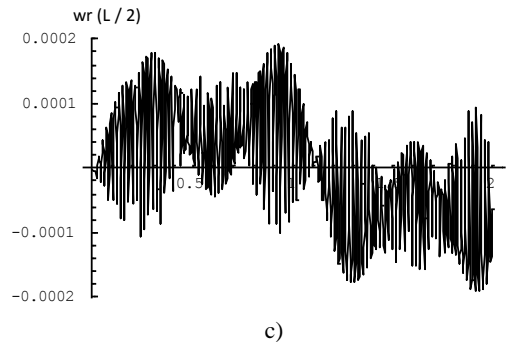


Figure 13. c) Deformation of the bridge due to roughness only

In the two cases studied above, we observe that the trouble caused by the roughness of the bridge is slightly less than that caused by the roughness of the road.

The above observations and their causes, must be studied in detail taking into account the influence of the length of the bridge, the size of the vehicle, its speed, etc. The above is the subject of a future paper by the authors.

## 5 CONCLUSIONS

Passengers' trouble can be expressed by different ways.

The most usual and easy way is to use either the width of the motion due to the roughness, or the acceleration of this motion due to roughness alone, which happens in low periods. In this paper we follow the first way.

From the results of the models considered, one can draw the following conclusions:

Regarding the roughness on road:

- For smooth road surface with  $d \leq 0.01 m$ , the induced trouble can be considered as practically non-existent
- For middle road surface  $0.01 \leq d \leq 0.02 m$ , the induced trouble is rather tolerable.
- For bad road surface  $d \geq 0.03 m$ , the induced trouble is remarkable.
- Finally for a very bad road surface  $d \geq 0.05 m$ , the trouble is extremely annoying.

Regarding the roughness on bridge:

We observe that the trouble caused by the roughness of the bridge is slightly less than that caused by the roughness of the road.

The above observations and their causes, must be studied in detail taking into account the influence of the length of the bridge, the size of the vehicle, its speed, etc. The above is the subject of a future paper by the authors.

## REFERENCES

- [1] Carey W.N. Jr., Irick P.E., "The pavement serviceability-performance", Concept. HRB, Bull. 250, 1960, 40-48
- [2] Kaneshige I. "Power spectra analysis of a vehicle vibration by Road Surface Roughness", J. of Society of Automotive Eng. of Japan, V. 18(4), 1964, 261-267.
- [3] Kawai T., Morisaki K., "Relationships between Characteristics and Road Surface and Riding Comfort of Vehicle", Mitsubishi Juko Giho, V. 2(2), 1965, 20-27.
- [4] Dodds G.J., Robson J.D., "Description of Road Surface Roughness", J. of Sound and Vibration, V. 31(2), 1973, 175-183.
- [5] Healey A.J., Nathman E., Smith C.C., "An Analytical and Experimental Study of Automobile Dynamics with Random Roadway Inputs", Journal of Dynamic Systems, Measurement and Control, Series G. Vol. 99, 1977, 284-292.
- [6] Honda H., Kajikawa Y., Kobori T., "Spectra of Road Surface Roughness on Bridges", J. of Structural Division (ASCE), V. 108, 1982, 1956-1966.
- [7] Wang T.L., Huang D., "Cable-Stayed Bridge vibration due to road surface roughness", Journal of Structural Engineering, 118(5), 1995, 1354-1374.
- [8] Cheng Y.S., Au F.T.K., Cheung Y.K., and Zheng D.Y., "On the Separation between Moving Vehicles and Bridge", J. of Sound and Vibration 222(50), 1999, 781-810.

- [9] Fatard M., Laflamme M., Savarad M., and Bennur M., “Dynamic Analysis of Existing Continuous Bridge”, *Journal of Bridge Engineering*, 3(1), 1998, 28-37.
- [10] Chang D., and Lee H., “Impact factors for simple-span highway girder bridges”, *Journal of Structural Engineering*, 120DeWolf(3), 1994, 704-715.
- [11] Huang D.P., and Wang T.L., “Vibration of highway steel bridges with longitudinal grades”, *Computers and Structures*, 69(2), 1998, 235-245.
- [12] Kou J.W., and DeWolf J.T., “Vibration behavior of continuous span highway bridge-influencing variables”, *Journal of Structural Engineering*, 123(3), 1997, 333-344.
- [13] Yang Y.B., Liao S.S., and Lin B.H., “Impact Formulas for Vehicles Moving over Simple and Continuous Beams”, *Journal of Structural Engineering*, 121(11), 1995, 1644-1650.
- [14] Wang P., Ding Yong., “Optimization of vehicle-induced vibration and structure-born noise of Bridge Based on computing”, *Int. J. of Engineering and Technology*, V. 11(3), 2019.
- [15] Wang G., Zhou J., “Research on the Influence of Pavement Roughness level on the Driving Comfort”, *Int. Conference on Civil and Hydraulic Engineering*, IOP Conf. Series: Earth and Environmental Sciences 304, 2019, doi: 10.1088/1755-1315/304/3/032059.
- [16] Koc M.A., Esen I., “Modeling and analysis of vehicle-structure road coupled interaction considering structural flexibility, vehicle parameters and road roughness”, *Journal of Mechanical Science and Technology*, 31(5), 2017, 2057-2074.
- [17] Park D.W., Papagiannakis A.T., Kim I.T., “Analysis of Dynamic Vehicle loads using Vehicle Pavement interaction model”, *KSCE Journal of Civil Engineering*, 18(7), 2014, 2085-2092.
- [18] Yang Y.B., Li Y.B., Chang K.C., “Effect of road surface roughness on the response of a moving vehicle for identification of Bridge frequencies”, *Interaction and Multiscale Mechanics*, V. 5(4), 2012, 347-368.
- [19] Kim C.W., Inone S., SURIURA K., “Extracting bridge frequencies from dynamic responses of two passing vehicles”, *Insights and Innovations in Structural Engineering Mechanics and Computation – Zingoni Ed.*, 2016, 1858-1860.
- [20] Lalthlamuana R., Talukdar S., “Conditions of Visibility of Bridge Natural Frequency in Vehicle Vertical Acceleration”, *Procedia Engineering* 144, 2016, 26-33.
- [21] LeNgo-Tran T., Hayashikawa T., Matsumoto T., “Three-dimensional bridge-vehicle interaction analysis of simply supported twin I-girder bridge”, *J. of Structural Engineering* V.54A, 2008, 181-188.
- [22] Wang T.L., Huang D., Shahawy M., “Vibration and Impact in Multigirder Steel Bridges”, *J. of Structural Engineering* V. 118(8), 1992, 2222-2238.
- [23] Kounadis A.N., “An efficient and simple approximate technique for solving nonlinear initial and boundary-value problems”, *Computational Mechanics*, 9, 1992, 221-231.